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<p>(54) Title: METHOD AND APPARATUS FOR HANGING TUBULARS IN WELLS</p> <p>(57) Abstract</p> <p>A well tubular (14) is hung within a casing (10) through placement of tubular on overlapping relationship with the casing. A spacer (16) may be located therebetween. The tubular (14) is expanded which in turn expands the spacer (16) when used. The tubular is expanded beyond the yield point such that it or an intervening spacer engages the inside of the casing (10) and stresses the casing within its elastic limits. The assembly then contracts to form a tight structural support between the tubulars and a high pressure seal against flow therebetween. A spacer (16) having channels (20, 22) about either end with ductile sealing material (24) therein is of an expanded metal material through cuts in the sheet. A hydraulic ram (32) is employed with an expandable collet (54) to draw the collet through the overlapping area of the liner and the casing. The collet extends within a shoulder (68) at the end of its stroke such that it will be substantially released from the upper end of the liner. A method for placing lateral liner includes expanding the liner to beyond its yield point within the hole through the casing. The stub of the liner positioned within the casing may then be drilled out such that completed lateral and main bores are achieved.</p> <img alt="Technical drawing of a well tubular assembly. The drawing shows a vertical cross-section of a well bore. A main tubular 14 is shown within a casing 10. A liner 12 is positioned within the tubular 14. A collet 54 is shown being expanded into the liner 12 and the casing 10. Various components like a hydraulic ram 32, a spacer 16, and a shoulder 68 are labeled. Reference numerals include 10, 12, 14, 16, 18, 20, 22, 24, 26, 30, 32, 54, 58, 68, 76, 82, 100, 102, 104, 106, 108, 110, 112, 114, 116, 118, 120, 122, 124, 126, 128, 130, 132, 134, 136, 138, 140, 142, 144, 146, 148, 150, 152, 154, 156, 158, 160, 162, 164, 166, 168, 170, 172, 174, 176, 178, 180, 182, 184, 186, 188, 190, 192, 194, 196, 198, 200, 202, 204, 206, 208, 210, 212, 214, 216, 218, 220, 222, 224, 226, 228, 230, 232, 234, 236, 238, 240, 242, 244, 246, 248, 250, 252, 254, 256, 258, 260, 262, 264, 266, 268, 270, 272, 274, 276, 278, 280, 282, 284, 286, 288, 290, 292, 294, 296, 298, 300, 302, 304, 306, 308, 310, 312, 314, 316, 318, 320, 322, 324, 326, 328, 330, 332, 334, 336, 338, 340, 342, 344, 346, 348, 350, 352, 354, 356, 358, 360, 362, 364, 366, 368, 370, 372, 374, 376, 378, 380, 382, 384, 386, 388, 390, 392, 394, 396, 398, 400, 402, 404, 406, 408, 410, 412, 414, 416, 418, 420, 422, 424, 426, 428, 430, 432, 434, 436, 438, 440, 442, 444, 446, 448, 450, 452, 454, 456, 458, 460, 462, 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DESCRIPTIONMETHOD AND APPARATUS FOR HANGING TUBULARS IN WELLS5 BACKGROUND OF THE INVENTION

The field of the present invention is well drilling and completion systems.

Well drilling and completion equipment includes tubulars which are variously characterized as casing, tubing and liner. For universal application, they are cylindrical in 10 shape and of a length in compliance with the American Petroleum Institute Standard 5C. The term "casing" is typically applied to tubulars which are larger in diameter and used to support the earth's encroachment when drilling a bore hole for a well. Often casing is

unwanted gases, water or other fluids outwardly of the casing. Casing is typically 15 assembled from 40 foot long tubulars with threaded couplings. Wells can extend for several miles into the earth. As the well increases in depth, the hydraulic pressures to which the casing is subjected to increase. Decreases in casing diameter with increasing depth is common, often to avoid experiencing excessive force from such high pressures. Such decreases typically occur in step function as smaller casing is employed.

20 "Liner" is typically made up of tubulars in an area of well production. Liner can have portions with slots prefabricated through the wall, end closure elements and the like. Liner is typically smaller in diameter than casing and is typically placed in wells after casing to extend from casing into production zones.

25 Other tubing may be employed within casing to bring production to the surface and for other communication within wells. This too is placed in wells after casing and has a reduced diameter.

To insure the flow of fluids with or without entrained solids are appropriately 30 directed within wells, packers or annular seals are frequently employed to span gaps at radial steps in tubular construction within wells. Packers are also employed to insure the blockage of pressure from unwanted areas.

Additionally, structural support from above frequently is needed for such placements. The compression of tubular strings through placement on the bottom is often considered to be detrimental to the pressure integrity of the structure. Consequently,

suspending liner or casing in tension is preferred. Hangers typically are used which employ wedges or other structural devices to grip the inner tubular. Combinations of packers and hangers are also used.

5 SUMMARY OF THE INVENTION

The present invention is directed to methods for hanging tubulars in wells including the expansion of the inner tubular beyond its elastic limit outwardly against an outer tubular with the outer tubular experiencing sufficient deformation to place the final assembly in a tight relationship. Tubular hanging is accomplished. Sealing may also be 10 achieved. Apparatus to these ends is separately contemplated.

In a first separate aspect of the present invention, a method for hanging an inner tubular and an outer tubular includes an overlapping of the tubulars. The inner tubular is

15 expanded partially or fully circumferentially by the inner tubular, the expansion being sufficient that elastic recovery for the inner tubular is less than elastic recovery for the outer tubular. A structural hanging of the inner tubular on the outer tubular is thus accomplished. Depending on the materials employed, a sealing may also be accomplished at the same time. Additional ductile sealing material may be employed as well. The foregoing can be accomplished without expanding the outer tubular beyond the yield point 20 when that is preferred.

In a second separate aspect of the present invention, a method for hanging a first tubular and a second tubular includes an overlapping of the tubulars with a spacer therebetween which is substantially incompressible in the radial direction. The inner tubular is expanded partially or fully circumferentially past the yield point and the outer tubular is expanded partially or fully circumferentially by the spacer. A structural hanging 25 of the inner tubular on the outer tubular is thus accomplished. Depending on the materials employed, a sealing may also be accomplished at the same time. The spacer may have seals and structure allowing for its easy partial or full expansion circumferentially through portions thereof. Additional ductile sealing material may be employed as well.

30 In a third separate aspect of the present invention, the prior aspects are contemplated to be specifically employed for hanging cylindrical liners within cylindrical casings.

In a fourth separate aspect of the present invention, laterally hanging a tubular is

accomplished through drilling diagonally through the wall of a casing, placing a tubular through that wall and expanding the tubular past the yield point and the casing by the tubular. The tubular extending into the casing may then be drilled out. In this way, access to the main bore as well as to the lateral bore or bores remains.

5 In a fifth separate aspect of the present invention, a spacer contemplated for use between tubulars of different diameters is contemplated. A tubular body includes inner and outer circumferential channels with ductile seals arranged therein. Longitudinal slits through the wall of the tubular body facilitate expansion of the spacer. The slits are staggered and do not extend to the circumferential channels.

10 In a sixth separate aspect of the present invention, a tubular expander includes a hydraulic ram with a shoulder and a draw bar extending through the shoulder. A collet is associated with the draw bar and cooperates with the draw bar through beveled surfaces to

15 ~~SECUREMENT~~
on the draw bar to control collet expansion. The shoulder on the hydraulic ram may also be extended to receive at least a portion of the collet such that the maximum diameter of the collet may be drawn substantially fully through the end of the tubular.

In a seventh separate aspect of the present invention, the tubular expander of the prior aspect is contemplated to be associated with a tubular with the collet expanded to firmly engage the tubular.

20 In an eighth separate aspect of the present invention, combinations of the foregoing aspects are contemplated.

Accordingly, it is an object of the present invention to provide hanging methods for wells and apparatus associated therewith. Other and further objects and advantages will appear hereinafter.

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BRIEF DESCRIPTION OF THE DRAWINGS

Figure 1 is a partial cross-sectional view of a tubular within a casing with a tubular expander.

30 Figure 2 is a partial cross-sectional view of a tubular within a casing expanded into hanging relationship therewith.

Figure 3 is a cross-sectional detail view of the wall of Figure 2 with an added seal layer.

Figure 4 is a spacer shown in partial cross section.

Figures 5A and 5B show a tubular expander illustrated in partial cross section.

Figures 6A-6H are a sequential schematic series of cross sections of a multi-lateral tubular placement.

5 DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Turning in detail to the drawings, Figure 1 illustrates a tubular, shown to be a casing 10 in this embodiment, understood to be positioned within a well bore (not shown). The lower end 12 of the casing 10 does not extend to the bottom of the well bore. An assembly for hanging a second tubular, shown to be a liner 14 in this embodiment, within the casing 10 is positioned with the liner 14 in an overlapping relationship with the casing 10. This second tubular may be casing, liner or other tubing with a smaller diameter than the first tubular with which it is positioned. The liner 14 extends further into the well an 10
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drilling stock which are conventional standard tubulars.

15 A spacer 16 may be located between the liner 14 and the casing 10. When a spacer 16 is used, it preferably extends to surround the area of the liner 14 which is overlapping with the casing 10 and which is to be expanded outwardly against the casing 10. A wide variety of spacers 16 may be employed. Separate spaced collars, a wrapping of substantially incompressible filler material and the like are contemplated. One such 20
25
spacer 16 is best illustrated in Figure 4.

The spacer includes a tubular body 18 with outer channels 20 near either end. Inner channels 22 are also near either end. Both channels 20 and 22 receive conventional sealing material 24 which is packed to extend in the uncompressed state outwardly from the channels 20 and 22.

25 The material of the tubular body 18 is to be substantially incompressible in the radial direction. In this regard, the material is preferably similar to that of the casing 10 and the liner 14. As the liner 14 expands, the spacer 16 is anticipated to transfer certain of the load outwardly into the casing 10. The substantially incompressible nature is that which is sufficient to accomplish an appropriate force transfer.

30 The tubular body 18 further has slits 26. These slits are longitudinally staggered such that angularly adjacent such slits 26 are displaced longitudinally as can be seen in Figure 4. The slits preferably do not extend longitudinally along great distances. C-shaped slits 26 are contemplated as specifically illustrated. The slits 26 act to create an

expandable metal structure which resists partial or full circumferential expansion substantially less than the tubular liner 14. Even so, radial incompressibility is not significantly compromised.

The slits 26 do not extend fully to the ends of the tubular body 18 or even so far as the channels 20 and 22. In this way, an annular closed collar is defined at each end. 5 Each collar will require additional force for expansion. The ductile sealing material 24 will easily expand partially or fully circumferentially within the channels 20 and 22.

A ductile sealing material which may be a polymeric substance or a ductile metal 10 filler material may overlay the liner or the pacer 16 when one is employed. One such ductile sealing layer 28 is illustrated in the detail of Figure 3. A similar sealing layer (not shown) may also or alternatively be employed where appropriate between the liner 14 and the spacer 16.

15 expander, generally designated 30, is shown in detail in Figures 5A and 5B and is shown in position before expansion in Figure 1.

The tubular expander 30 includes a hydraulic ram 32 which includes a cylinder 34 having ram annular pistons 36 and 37. A draw bar 38 is positioned inwardly of the cylinder 34. The draw bar 38 has a central bore 40 which may be closed at the distal end thereof by a cap 42 or other means such as additional equipment further down hole. The 20 draw bar 38 includes shoulders 44 and 46 which, with the bar itself, the cylinder 34 and the ram annular pistons 36 and 37 define ram expansion spaces 48 and 50, respectively. Lip seals or O-rings are appropriately positioned to ensure sealing of the ram expansion spaces 48 and 50. The shoulder 46 is shown to be a separate element rather than integral as is shoulder 44. This is appropriate for ease of assembly. Further, additional shoulders 25 46 may be associated with additional ram annular pistons 36 and 37 where more force is necessary. Passages 52 are shown to extend from the central bore 40 to the ram expansion spaces 48 and 50 for the delivery of high pressure fluid. Relief passages 53 avoid pressure buildup behind the piston 37 as the hydraulic ram 32 moves through its stroke.

Depending upon the pressure which may be necessary for expanding a tubular, 30 not only may force advantage be achieved through the multiplication of ram annular pistons 36 but a hydraulic intensifier may be employed above the tubular expander 30. The principles of hydraulic intensifiers are well known as requiring a small input piston capable of traveling through a relatively large distance and driving a larger output piston

capable of traveling through a much shorter distance and exerting a far higher force. The hydraulic force generated by the larger piston would then be input into the central bore 40 for distribution through the passages 52 into the ram expansion spaces 48 and 50.

The draw bar 38 extends from the cylinder 34 and receives a collet, generally 5 designated 54. The collet 54 includes a ring 56 at its lower end formed in two portions for ease of manufacture. Segments 58 extend from the ring 56 about the draw bar 38 and toward the hydraulic ram 32. These segments 58 are cantilevered from the ring 56 such that they may be forced to expand outwardly from a retracted force neutral position. Slots 60 define the segments 58 and are shown to include a jog at the thickest portion of the 10 collet 54 so as to provide continuous expansion force about the entire collet.

The draw bar 38 includes a beveled outer surface portion 62 and an outer shoulder 64 which extend fully about the draw bar 38. Each segment 58 similarly includes a beveled inner surface portion 66 which extends fully about the draw bar 38. As can be seen from Figures 5A and 5B, as the collet 54 moves 15 downwardly relative to the draw bar 38, the beveled outer surface portion 62 and the beveled inner surface portion 66 act together to expand the segments 58 outwardly in a radial direction. The outer shoulder 64 and the inner shoulder 68 cooperate to limit the relative travel between the collet 54 and drawbar 38 so as to limit the expansion of the collet.

To effect the foregoing relative longitudinal displacement of the collet 54 on the 20 draw bar 38, an annular piston 70 associated with the ring 56 of the collet 54 cooperates with the draw bar 38 to define an expansion space 72. A further passage 74 extends from the central bore 40 to the expansion space 72. Seals about the expansion space 72 inhibit leakage. Thus, the pressure commencing to draw the hydraulic ram 32 upwardly also 25 drives the collet 54 downwardly to expand the segments 58.

A retaining ring 76 located at the distal end of the segments 58 is affixed to the draw bar 38. This ring 76 includes a first cavity 78 to retain the ends of the segments 58 when in the contracted state as illustrated in Figures 5A and 5B and a second cavity 80 to retain the ends of the segments 58 when in the expanded state.

30 Referring back to the cylinder 34 of the hydraulic ram 32, a shoulder 82 is located at the lower end of the cylinder 34 and displaced therefrom. The draw bar 38 extends through this shoulder 82. The extension of the shoulder 82 is of sufficient length and inner diameter such that it can receive the upper end of the collet 54 and the retainer ring 76.

The extension of the shoulder 82 is to the maximum diameter of the collet 54 when in the expanded state. Extraction of the tubular expander assembly once drawn through the full stroke is thereby accomplished without further tubular expansion of the liner 14.

In operation, a smaller diameter tubular, such as the liner 14, selected to be placed 5 within a larger diameter tubular, such as the casing 10, already in position within a well. A spacer 16 may first be positioned about the liner 14 adjacent one end, particularly if the necessary expansion of the liner 14 would otherwise be excessive. The spacer or spacer elements are selected to extend substantially the length of the portion of the liner 14 to be expanded. Ductile sealing material may be added about the liner. Where a spacer is 10 present, such ductile sealing material may be either inwardly of the spacer 16 or outwardly of the spacer 16 or both.

Once the tubular has been prepared, a tubular expander is placed therein. A tubular collet 54 is selected with an appropriate piston stroke to expand a portion of the length of the liner 14. The draw bar 38 is extended such that the widest area of the collet 15 54 is in location to expand the desired portion of the liner 14. With a spacer involved, the collet is arranged just longitudinally outwardly of the spacer 16. With the appropriate length selected, the shoulder 82 on the hydraulic ram 32 abuts against the near end of the liner 14. Some pressure may be supplied to the central bore 40 so as to set the collet 54 within the liner 14 with enough force so that the entire liner assembly can be supported by 20 the collet 54 as the assembly is lowered into the well.

Once in position with the liner 14 overlapping the casing 10 at least to the extent of the spacer 16, high pressure fluid is directed down the drill pipe to the central bore 40 of the draw bar 38. This pressure acts to drive the collet 54 on the draw bar 38 to the fully expanded position. The pressure also acts to draw the expanded collet 54 upwardly 25 through the liner 14 toward the shoulder 82 of the hydraulic ram 32.

The inner diameter of the casing 10 and the outer diameter of the liner 14 are selected along with the appropriate thickness of the spacer 16, if used, such that operation of the collet 54 being drawn through the portion of the liner 14 will expand the liner which in turn expands the spacer 16. The expansion of the liner 14 is beyond the yield point of 30 the material. In this way the gap necessary for placement, either between the liner 14 and the casing 10 or the spacer 16 and the casing 10, is permanently closed. The yield point of any material is determined by convention, typically at .2% offset yield. Because of the necessary gap, significant plastic strain beyond the yield point is anticipated.

Either the liner 14 itself or the spacer 16 extends outwardly to expand the casing 10. The assembly is preferably but not necessarily selected such that the expansion of the casing 10 remains within the elastic limit of the material. The elastic expansion of the casing 10 is such that, with the tubular expander withdrawn, the casing 10 is able to rebound enough to remain tight against the liner 14 or the spacer 16 and in turn the liner 14. Further, it is commonly understood that the materials of oil field tubulars are able to be stretched in the yield range to as much as about 10% to 20% or more without experiencing a significant decrease in strength. Competing effects of work hardening and reduction in cross-section accompanying the inelastic strain results. With continued expansion, the reduction in cross section becomes the dominant factor and strength decreases. The strength of concern is typically the longitudinal tensile strength of the tubular.

When expanded, the inner tubular expands more than the outer tubular per unit of circumference. Likewise, when recovering after the load is removed, the inner tubular will shrink less than the outer tubular to achieve the same ratio of recovery. Consequently, the outer tubular will remain in some tension and the inner tubular will remain in some compression if the two are expanded with the inner tubular expanding in excess of the yield point enough so that the inner tubular cannot recover to a position where tension is removed from the outer tubular. In other words, the outer tubular may remain within the elastic limit but is preferably expanded enough so that its recovery when unloaded by the tubular expander is at least as great as the recovery of the inner tubular. A minimum expansion of both tubulars is preferred to achieve this result. Expansion to the point that a tubular begins to lose strength is avoided except in unusual applications.

Once the collet 54 has been drawn as far as possible through the shoulder 82 by the draw bar 38, it is substantially free from the now expanded liner portion 14. With this accomplished, the drill string with the collet 54 attached can be withdrawn from the well. If other elements are located below the collet 54 on the drill string, they may be employed for gravel packing, cementing and the like.

Turning to the method of laterally hanging a tubular as sequentially illustrated in Figures 6A-6H, a first trip down the well with the liner in place includes a whipstock 84 of conventional design in association with a drill in liner 86 typically employing a mud motor and geosteering. In Figure 6A, the whipstock is being placed. In Figure 6B the whipstock 84 is now set and disengaged from the drill in liner 86. In Figure 6C, the drill in liner is

shown cutting a window or hole through the casing. The drilling continues until the drill in liner 86 has almost completely passed through the window in the casing. A tubular expander was included as part of the drill in liner assembly. Once the drill in liner 86 has been placed, the collet is opened and drawn through the liner 86 across the window in the casing. The liner 86 expands and becomes fixed within the window of the casing. The attachments are then withdrawn, leaving the drill in liner 86 in place.

In Figure 6F, a drill is shown being positioned down the well on a second trip to take out the stub of the drill in liner 86 which extends into the interior of the casing. The whipstock is then attached and withdrawn leaving a completed lateral bore and a completed main bore with full bore access. The lateral liner is mechanically connected and provides a high pressure seal.

Accordingly, improved methods and apparatus are disclosed for the hanging of tubulars within a well. While embodiment and application of the invention have been shown and described, it would be apparent to those skilled in the art that many more modifications are possible without departing from the inventive concepts herein. The invention, therefore is not to be restricted except in the spirit of the appended claims.

CLAIMS:

1. A method for hanging a first tubular in a second tubular positioned in a well, comprising

5 placing the first tubular within the second tubular in an overlapping relationship with the first tubular extending into the well from the second tubular;

holding the first tubular in place;

expanding at least a portion of the first tubular overlapping with the second tubular partially or fully circumferentially past the yield point including expanding the

10 second tubular partially or fully circumferentially through expansion of the first tubular, the expanding being sufficient that elastic recovery for the first tubular is less than elastic recovery for the second tubular.

2. The method of claim 1, the second tubular being expanded within the
15 elastic limit of the second tubular.

3. The method of claim 1 further comprising
surrounding the portion of the first tubular with ductile sealing material.

20 4. The method of claim 1, expanding the first tubular being in the range of
increased strength of the first tubular.

5. The method of claim 1 further comprising
surrounding a portion of the first tubular with a spacer which is substantially
25 incompressible in a radial direction of the first tubular, expanding at least a portion of the
first tubular including expanding the spacer.

6. The method of claim 5 further comprising
surrounding the portion of the liner and the spacer with ductile sealing material.

7. The method of claim 1 further comprising
drilling diagonally through the wall of the second tubular, placing the first tubular
within the second tubular including placing the first tubular through the wall of the second

tubular in an overlapping relationship with the second tubular with the first tubular extending diagonally from the second tubular into the lateral well;

drilling out the first tubular extending within the second tubular.

5 8. The method of claim 7, the second tubular being expanded within the elastic limit thereof.

9. The method of claim 7, the expansion being sufficient that elastic recovery for the first tubular is less than elastic recovery for the second tubular.

10 10. The method of claim 7, expanding the first tubular being in the range of increased strength of the first tubular.

11. A tubular expander comprising
15 a hydraulic ram including a cylinder with a shoulder at one end and a draw bar extending through the shoulder, the draw bar having a beveled outer surface portion and an outer shoulder;

20 a collet about the draw bar including a ring and segments cantilevered from the ring along the draw bar toward the hydraulic ram, the segments each having a beveled inner surface portion slidably mating with the beveled outer surface portion to radially expand the collet with axial movement of the collet relative to the draw bar toward the cylinder and an inner shoulder facing the outer shoulder of the draw bar to limit movement of the collet relative to the draw bar, the shoulder of the cylinder extending beyond the cylinder to receive at least a portion of the collet therein.

25 12. The tubular expander of claim 11 further comprising
a tubular abutting against the shoulder of the cylinder and extending over the collet.

30 13. The tubular expander of claim 12 further comprising
a spacer surrounding the tubular and being substantially incompressible in a radial direction of the tubular.

14. The tubular expander of claim 13 further comprising
ductile seals, the spacer including a tubular body having inner and outer
circumferential channels near each end of the tubular body and longitudinal slits through
the wall of the tubular body, angularly adjacent ones of the longitudinal slits being
5 staggered longitudinally and not extending to any of the circumferential channels, the
ductile seals being in the inner and outer channels and extending outwardly from each
channel.

15. The tubular expander of claim 11, the annular piston, the draw bar and the
10 ring defining an expansion space adjacent the annular piston.

16. The tubular expander of claim 15, the cylinder including ram annular
pistons and ram expansion spaces defined by the draw bar and the cylinder adjacent the
ram annular pistons, respectively.

15
17. The tubular expander of claim 16, the draw bar having a center bore
closed at the distal end and passages to the expansion space and the ram expansion spaces.

18. The tubular expander of claim 17 further comprising
20 a retainer ring fixed to the draw bar and extending over the distal ends of the
segments of the collet, the retainer ring having a first cavity to receive the ends when the
collet is contracted and a second cavity to receive the ends when the collet is expanded.

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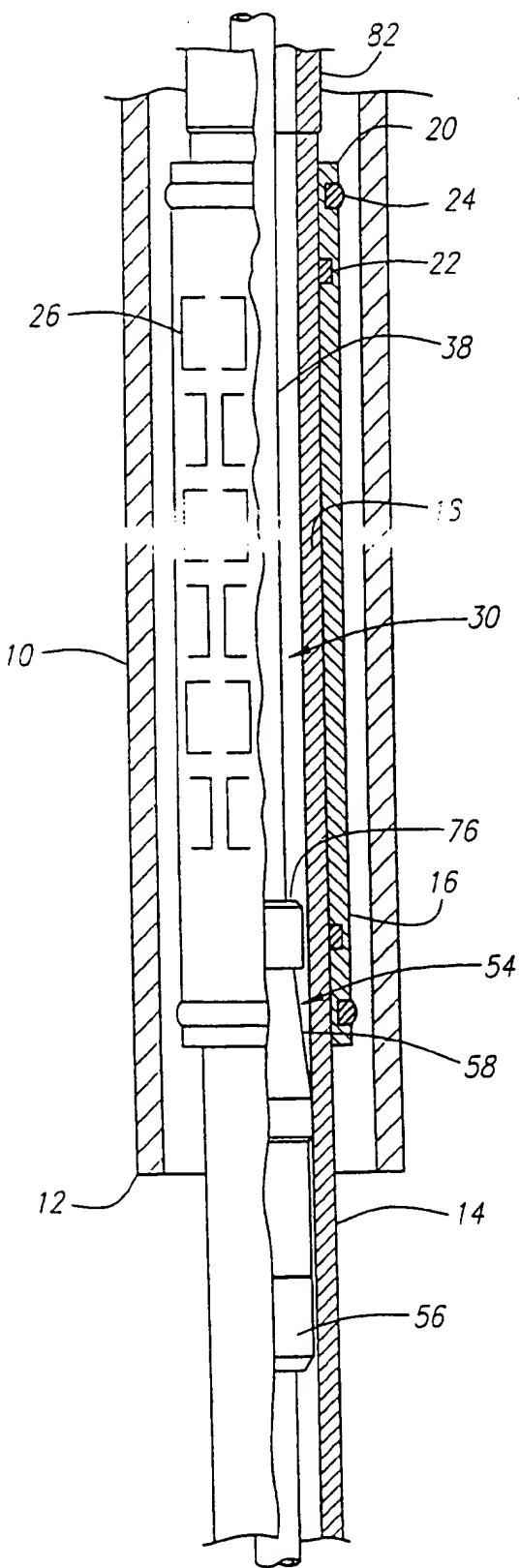


FIG. 1

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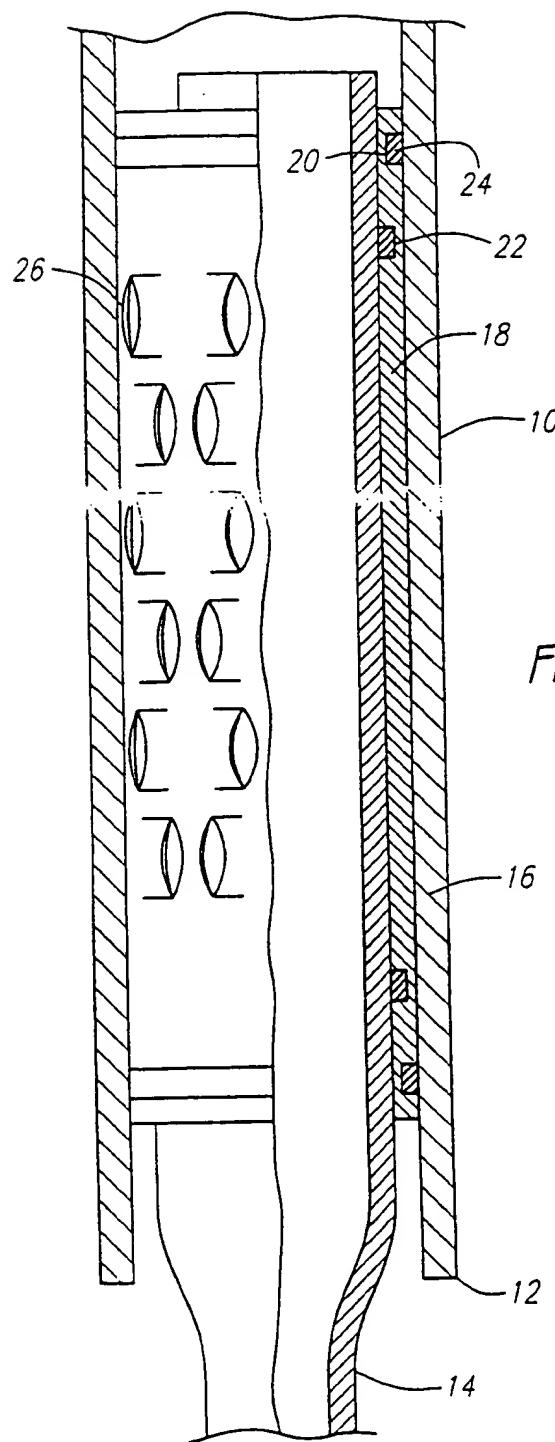


FIG. 2

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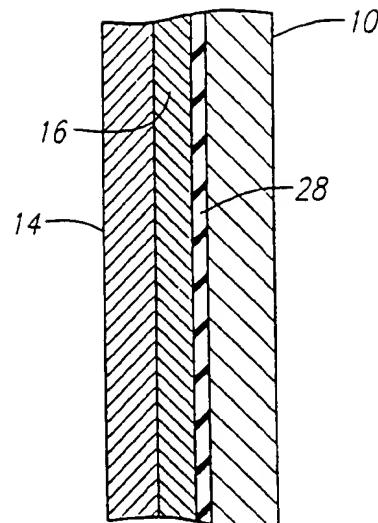


FIG. 3

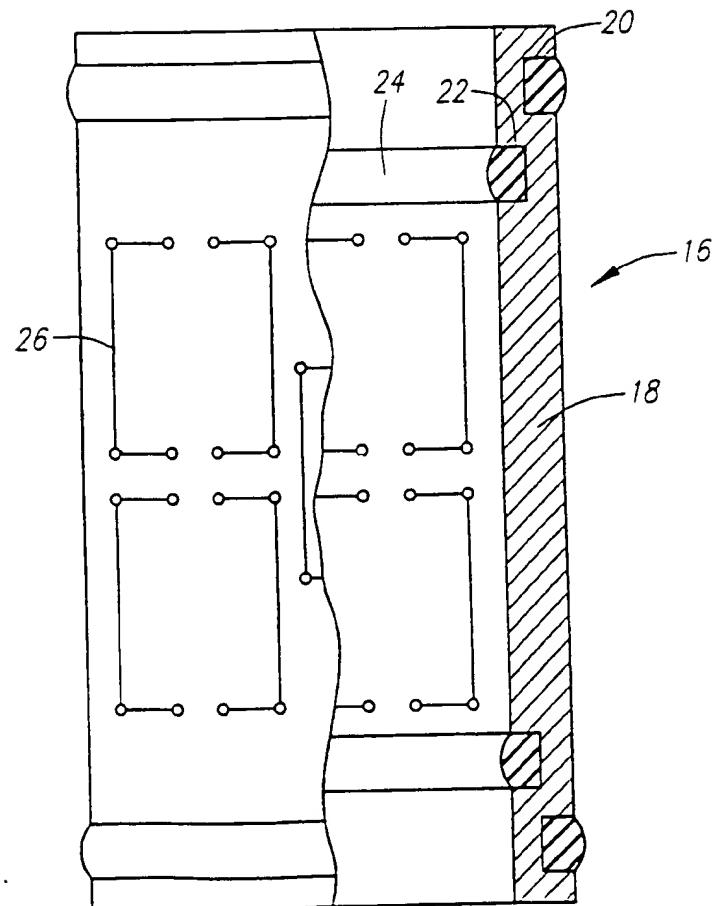
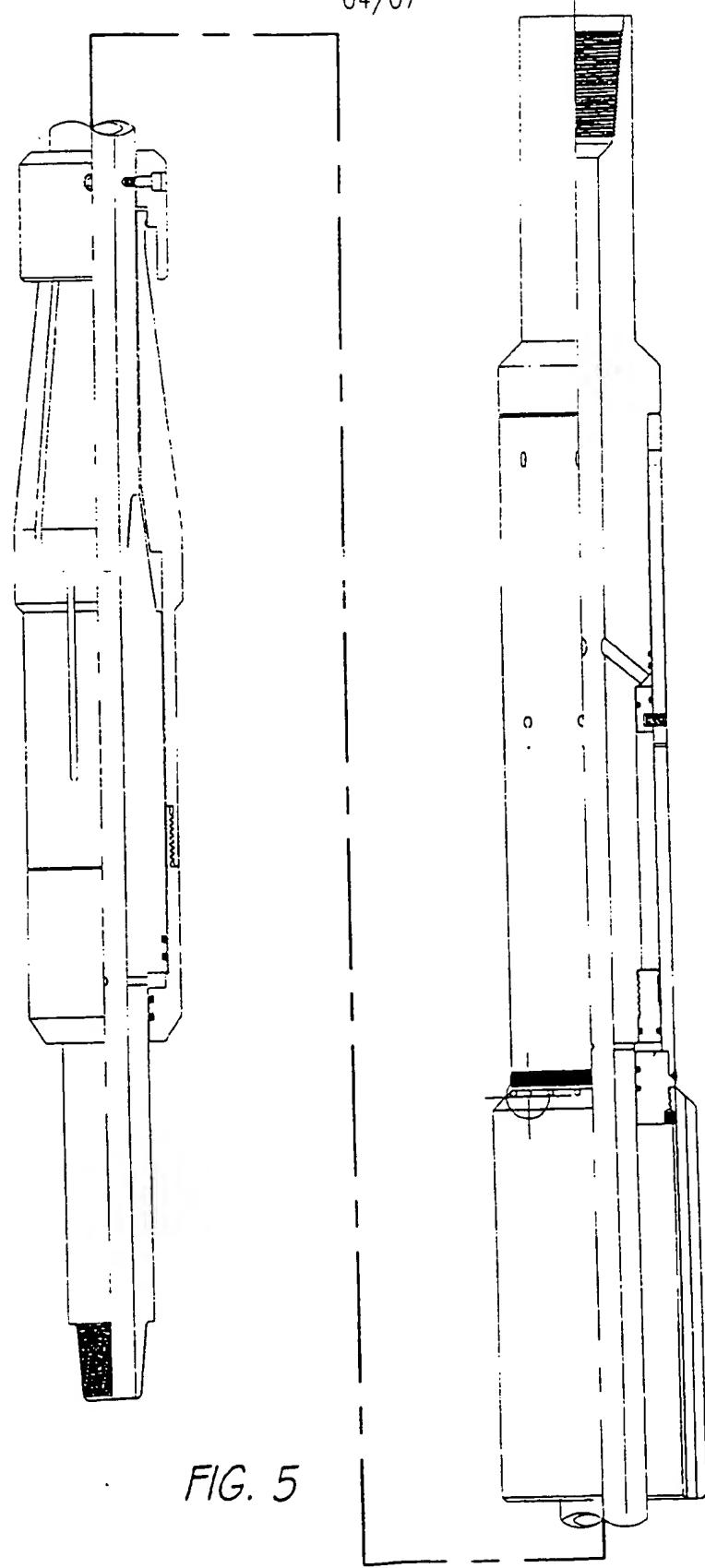


FIG. 4

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05/07

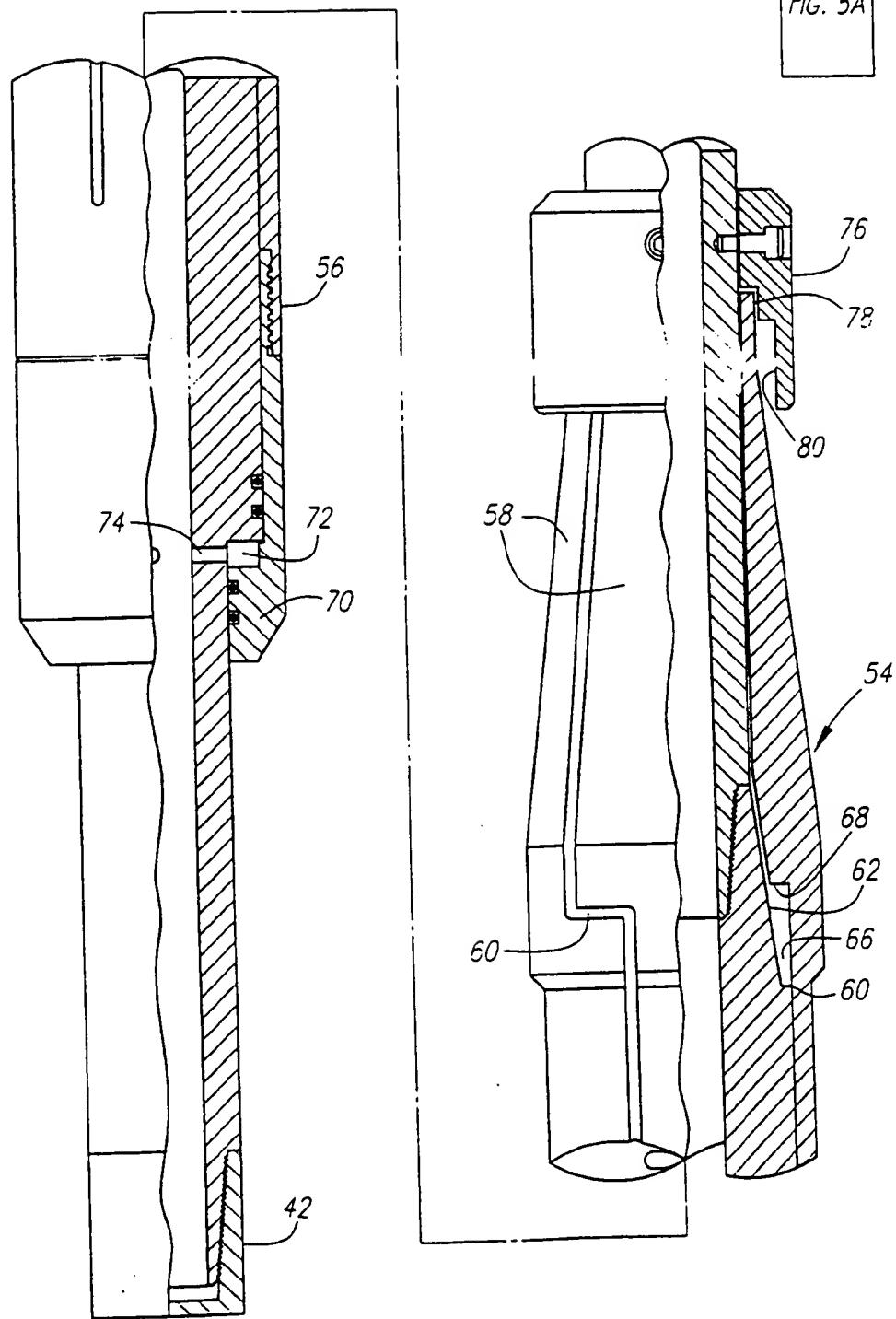
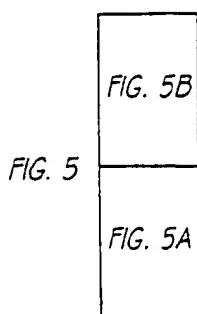


FIG. 5A

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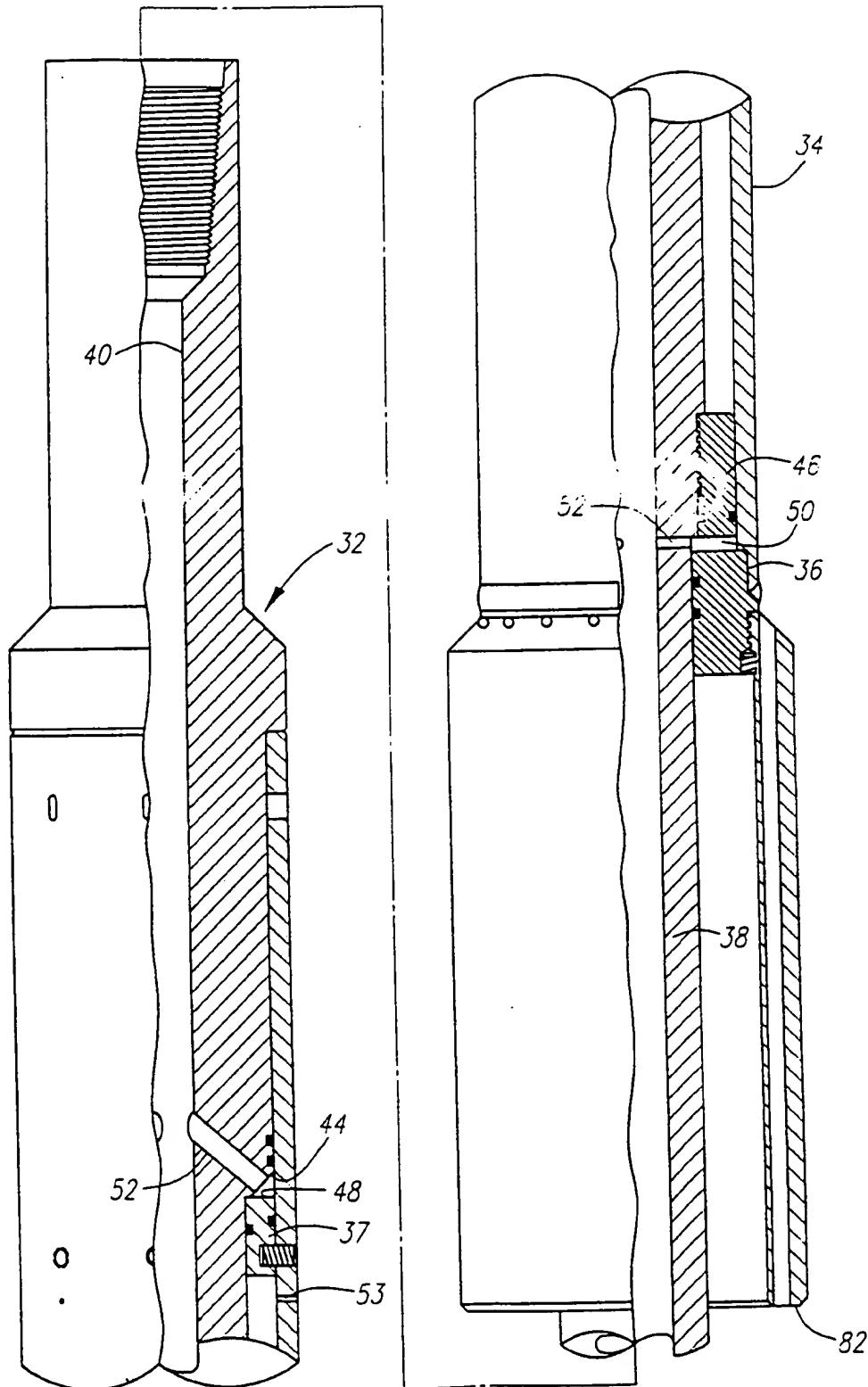
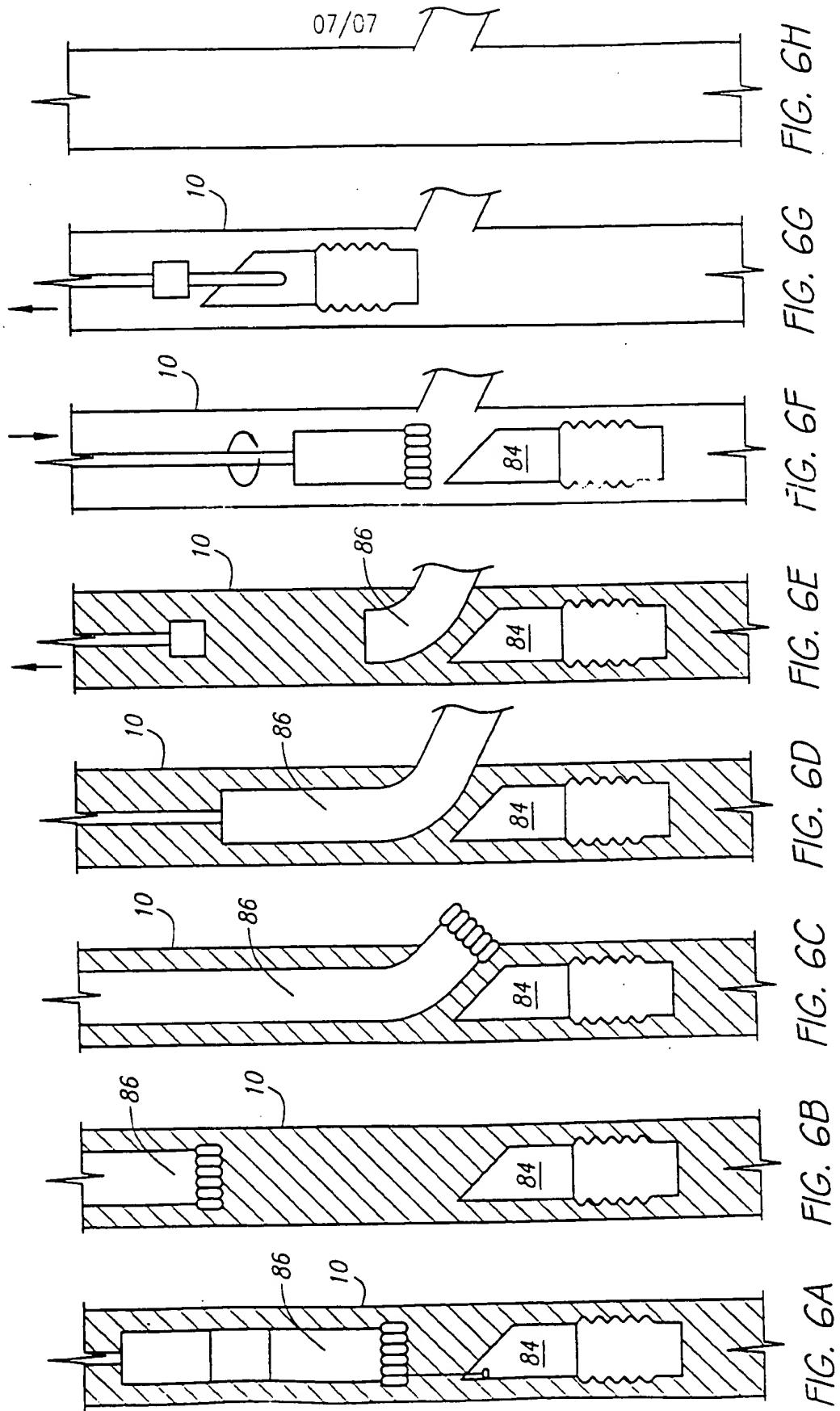


FIG. 5B



INTERNATIONAL SEARCH REPORT

International application No.
PCT/US98/21118

A. CLASSIFICATION OF SUBJECT MATTER

IPC(6) :E21B 23/01, 43/10

US CL :166/277, 207

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

U.S. : 166/277, 207, 206, 122, 187, 217

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

APS

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 2,214,226 A (ENGLISH) 10 September 1940 (10/09/40), Figs. 5-7, and entire document.	1, 2, 4
X	US 3,948,321 A (OWEN ET AL.) 06 April 1976 (06/04/76), Figs 1-5, col. 3, line 13- col. 11, line 55.	1, 2, 4
Y		11-13
X	US 5,366,012 A (LOHBECK) 22 November 1994 (22/11/94), Figs 1 and 2.	1, 2, 4
A	US 4,359,889 A (KELLY) 23 November 1982 (23/11/82), entire document.	1-18
A	US 3,583,200 A (CVILJANOVIC) 08 June 1969 (08/06/69), entire document.	1-18

 Further documents are listed in the continuation of Box C. See patent family annex.

•	Special categories of cited documents:	*T*	later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
•A•	document defining the general state of the art which is not considered to be of particular relevance	*X*	document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone
•B•	earlier document published on or after the international filing date	*Y*	document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art
•L•	document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reasons (as specified)	*A*	document member of the same patent family
•O•	document referring to an oral disclosure, use, exhibition or other means		
•P•	document published prior to the international filing date but later than the priority date claimed		

Date of the actual completion of the international search

14 FEBRUARY 1999

Date of mailing of the international search report

15 MAR 1999

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